(4 hours)

Q. P. Code: 18462

Maximum marks: 80

	N	I.B.	-	
1.		Question No.1 is compulsory.		
2.		Attempt any three out of remaining four questions.		
3.		Assume any suitable data if necessary and indicate it clearly.		
4.		Figures to the right indicate marks.		
5.		lustrate answers with sketches wherever required.	6	
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1.	Wr	ite short notes on any four.	20	
	a)	Gaskets and their selection.		
	b)	Various theories of failure.		
	c)	Design of stuffing box.		
	d)	Equipment classification.		
	e)	Various metal forming techniques.		
2.	a)	Describe design procedure of a flanged joint.	10	
	b)	Design the thickness of a cylindrical shell subjected to an internal pressure of	10	
	U	0.5 N/mm ² . Also check the resultant stress in the shell for safety with the	10	
		following data.		
		Permissible stress of the material used = 140 N/mm ²		
1		Internal diameter of the vessel = 1500 mm		
		Spot radiographically tested double welded butt joint ($J = 0.85$)		
		Total weight of the vessel and its content = 40 kN		
		Torque due to offset piping in the shell = 1500 Nm		
3.	a)	A cylindrical storage vessel has the following dimensions:	14	
٥.	4)	Diameter of tank = 12 m	****	
		Height of tank = 15m		
		Specific gravity of liquid to be stored = 1.2		
		Permissible stress of material of construction of tank = 98 N/mm ²		
		Joint efficiency factor = 0.85		
		Corrosion allowance = 1.5 mm		
		Modulus of elasticity = $2 \times 10^5 \text{ N/mm}^2$		
		Assume superimposed load of 1225 N/m ²		
		Density of steel = 7800 Kg/m ³		
		Design:		
		(a) i. Shell ii. Bottom iii Roof		
	b)	Draw to a recommended scale the front view of the shell and show the	6	
	.,	fabrication details.		
4	a)	Write a design procedure for agitator vessel which includes:	15	
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- i) Agitator shaft, ii) Blade assembly, iii) Stuffing box.
- b) Draw a proportionate drawing of stuffing box.

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5 a) Design a reaction vessel with following data.

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Inside diameter of shell = 1200 mm;

Jacket shell inside diameter = 1325 mm

Jacketed on straight side only with jacket length = 1000 mm

Design pressure shell = 0.4 N/mm^2

Design pressure Jacket= 0.45 N/mm²;

Design temperature (shell and jacket) = 150°C

Standard torisperical heads are provided on top and bottom.

Allowable stress for shell and head material = 120 N/mm²

Modulus of elasticity = $1.7 \times 10^5 \text{ N/mm}^2$

Poisson's ratio = 0.3

Allowable stress for jacket material = 95 N/mm²

Corrosion allowance for shell = 1.5 mm

Corrosion allowance for jacket = 2 mm

Internal diameter of half jacket coil = 100 mm

Weld joint efficiency factor for shell and jacket = 0.85

Weld joint efficiency factor for heads = 1

Factor B for (L/D_o with 0.825 and D_o/t = 202 considering thickness of shell as 6 mm) = 9000

If two stiffening rings are provided, Factor B for (L/D_0 with 0.275 and $D_0/t = 202$ considering thickness of shell as 6 mm) = 13000

Use stiffener of 40 mm x 40 mm (width x thickness)

b) Draw front view of reaction vessel.

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- 6. Write short notes on any four.
 - a) Classification of reaction vessels.
 - b) Saddle supports.
 - c) Skirt supports.
 - d) Wind girders.
 - e) Explain in detail metal joining arc welding method with neat sketch.